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GENERAL GUIDELINES

1. Working Plan:
 - a) Daily work assignments.
 - b) Work areas.
2. Materials
3. General Plan:
 - a) Materially available during
 - b) Estimating
 - c) Inventory list / Material order
 - d) Scheduling
 - e) Forwarding ahead the plan.
 - f) Temporary site office.
4. Shop-drawings and approvals.

PRESSURE VESSEL HANDBOOK

Twelfth Edition

with foreword by
Paul Butted
Professor of Chemical Engineering
University of Tulsa
Tulsa, Oklahoma

Eugene F. Megyesy

PRESSURE VESSEL HANDBOOK

Ninth Edition

with foreword by
PAUL BUTHOD
Professor of Chemical Engineering
University of Tulsa

EUGENE F. MEGYESY

PRESSURE VESSEL HANDBOOK
PUBLISHING INC.
P.O. Box 35365, Tulsa, OK 74153

EXTERNAL PRESSURE FORMULAE	
NOTATION	
P	External design pressure, psig
P_w	Maximum allowable working pressure, psig
P_c	Minimum design pressure, psig
P_s	Chosen value of sphere or hemispherical head, 4.50, for elliptical heads, both cover value of flange and elliptical heads, in.
R	Radius of spherical or hemispherical head, in.
t	Minimum thickness of section, in. (page 47)

SPHERE and HEMISPHERICAL HEAD	
The maximum allowable pressure: $P_w = \frac{2St}{R}$	
The value of t shall be determined by the following procedure:	
1. Assume the value for t and calculate the value of P_w from the formula on page 47 (page 47).	
2. Enter the appropriate material stress (page 47) at the value of P_w from the table of the appropriate material.	
3. From step 2, t .	
4. From step 1, P_w .	
5. Repeat steps 1 and 2 until the value of P_w is within 1% of the value of P .	
6. The maximum allowable working pressure, P_w , is the value of P_w obtained in step 5.	

2:1 ELLIPSOIDAL HEAD	
The required thickness shall be the greater of the following two values:	
(1) The thickness required by the formulae given for internal pressure on page 47 (page 47) using the external pressure and joint efficiency $E = 1.0$.	
(2) The thickness required by formulae $P_w = \frac{2St}{R(1 + \frac{2}{3} \frac{R}{t})}$, where R , t , and S to be determined as for spheres.	

4:1 ELLIPSOIDAL HEAD	
The required thickness and maximum allowable pressure shall be computed by the procedure given for elliptical heads. Use $sh = 0.8$, $sh_{max} = 0.8$.	

EXAMPLES	
DESIGN DATA:	
$P = 15$ psig external design pressure	
CS - 1020 carbon steel, 1020 steel	
Material of the head SA-516 steel	
300°F design temperature	

SPHERE	
Determine the required head thickness.	
SEE DESIGN DATA ABOVE	
Assume a head thickness: $t = 0.25$ in. $R_w = 48.00$ in.	
$t = 0.25000000$ in. $R_w = 48.00$ in.	
From Fig. 14-20-2.1 (page 42) $R_w = 48.00$ in. $R_w = 48.00$ in.	
From Fig. 14-20-2.1 (page 42) $R_w = 48.00$ in. $R_w = 48.00$ in.	
$P_w = 48000.000$ (psig) $= 48.00$ psig.	
Since the maximum allowable working pressure, P_w , is considerably greater than the design pressure, P , a more thickness would be satisfactory.	
For a second trial, assume a head thickness: $t = 0.1875$ in.	
$t = 0.1875$ in.	
$P_w = 64000.000$ (psig) $= 64.00$ psig.	
$P = 15$ psig, then $t = 0.1875$ in. $P_w = 64.00$ psig $= 64.00$ psig.	
The assumed thickness: $t = 0.1875$ in. is satisfactory.	

2:1 ELLIPSOIDAL HEAD	
SEE DESIGN DATA ABOVE	
Assume a head thickness: $t = 0.25$ in. $R_w = 48.00$ in. $R_w = 48.00$ in.	
$t = 0.25000000$ in. $R_w = 48.00$ in. $R_w = 48.00$ in.	
From Fig. 14-20-2.1 (page 42) $R_w = 48.00$ in. $R_w = 48.00$ in.	
From Fig. 14-20-2.1 (page 42) $R_w = 48.00$ in. $R_w = 48.00$ in.	
$P_w = 48000.000$ (psig) $= 48.00$ psig.	
Since the maximum allowable pressure, P_w , is greater than the design pressure, P , the assumed thickness is satisfactory.	

4:1 ELLIPSOIDAL HEAD	
SEE DESIGN DATA ABOVE	
Assume a head thickness: $t = 0.25$ in. $R_w = 48.00$ in. $R_w = 48.00$ in.	
$t = 0.25000000$ in. $R_w = 48.00$ in. $R_w = 48.00$ in.	
From Fig. 14-20-2.1 (page 42) $R_w = 48.00$ in. $R_w = 48.00$ in.	
From Fig. 14-20-2.1 (page 42) $R_w = 48.00$ in. $R_w = 48.00$ in.	
$P_w = 48000.000$ (psig) $= 48.00$ psig.	
Since the maximum allowable pressure, P_w , is greater than the design pressure, P , the assumed thickness is satisfactory.	

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PREFAC PARTICOG PART 1: DESIGN AND CONSTRUCTION OF VEGETS A Pressure Chapter 1: See them under internal pressure Chapter 2: VESORES UNDER EXTERNAL PRESSEN from nozzle 7: Reinforcement in the cone union to the Capätulo 8 cylinder: Welding of the vessels to chapter 9: regulations, specifications Chapter 10: Foreign Paigmeso Materials Chapter 11: Soldiers Capätulo 12: Capäas de tuber. 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In a mixture of gases, each constituent gas has a partial pressure which is the notional pressure of that constituent gas as if it alone occupied the entire volume of the original mixture at the same temperature. The total pressure of an ideal gas mixture is the sum of the partial pressures of the gases in the mixture (Dalton's Law). The partial pressure of a gas is a measure of ... Coronavirus - Service und Informationen Die Corona-Pandemie bedeutet drastische Einschnitte in allen Lebensbereichen. Auf dieser Seite finden Sie alle Informationen der Deutschen Rentenversicherung, die jetzt wichtig sind: Beratung und Erreichbarkeit, Online-Antragstellung, Servicetipps und vieles mehr: Drowning is a type of suffocation induced by the submersion or immersion of the mouth and nose in a liquid. Most instances of fatal drowning occur alone or in situations where others present are either unaware of the victim's situation or unable to offer assistance. 19/03/2021 - 日本ゴム協会東海支部講演会, 2017年11月16日, 名古屋. 山口政之 分子運動を利用した自己修復性高分子の設計と課題; 高分子学会北陸支部研究発表会, 2017年11月18日-11月19日, 新潟. 猪俣俊樹, 山口政之 非相溶系ポリマー-対における可塑剤の相間移動とその応用

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